

## Dynamic absorber

## Prof. Hugo Policarpo

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https://www.youtube.com/watch?v=WjePA0a8e\_c





### > Why we need/what to isolate/attenuate vibrations?

### It is not always the case!

Vibrating tables

**Rim polishing** 







## Why we need/what to isolate/attenuate vibrations?

- > Structural;
- > Dynamic stability;
- > Lifetime;
- Comfort;



## Structural





http://www.pbs.org/wgbh/nova/next/tech/rubber-bearings-seismic-protection/



https://vibrationdata.wordpress.com/



http://www.pardo.net/bike/pic/fail-001/FAIL-151.html



https://www.fos4x.de/en/applications-optimization-of-wind-turbines/

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https://www.youtube.com/watch?v=kX9dIxAyu0I

## **Dynamic stability**

#### Milestones in Flight History Dryden Flight Research Center



### PA-30 Twin Commanche Tail Flutter Test April 5, 1966

https://www.youtube.com/watch?v=iTFZNrTYp3k



https://www.youtube.com/watch?v=IXyG68\_caV4





https://www.youtube.com/watch?v=fWQwtBhIR\_0

## Lifetime and comfort



https://www.youtube.com/watch?v=hALQ-jqPV-Y

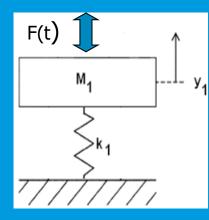


https://www.youtube.com/watch?v=YAGPyKNZhHw

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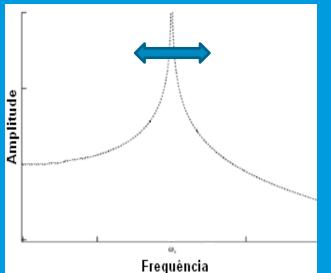




### Methods to "attenuate" vibrations



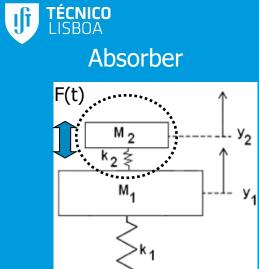






https://www.youtube.com/watch?v=rnhDpuq6Fgo

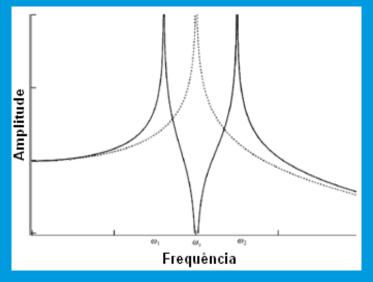
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### Methods to "attenuate" vibrations







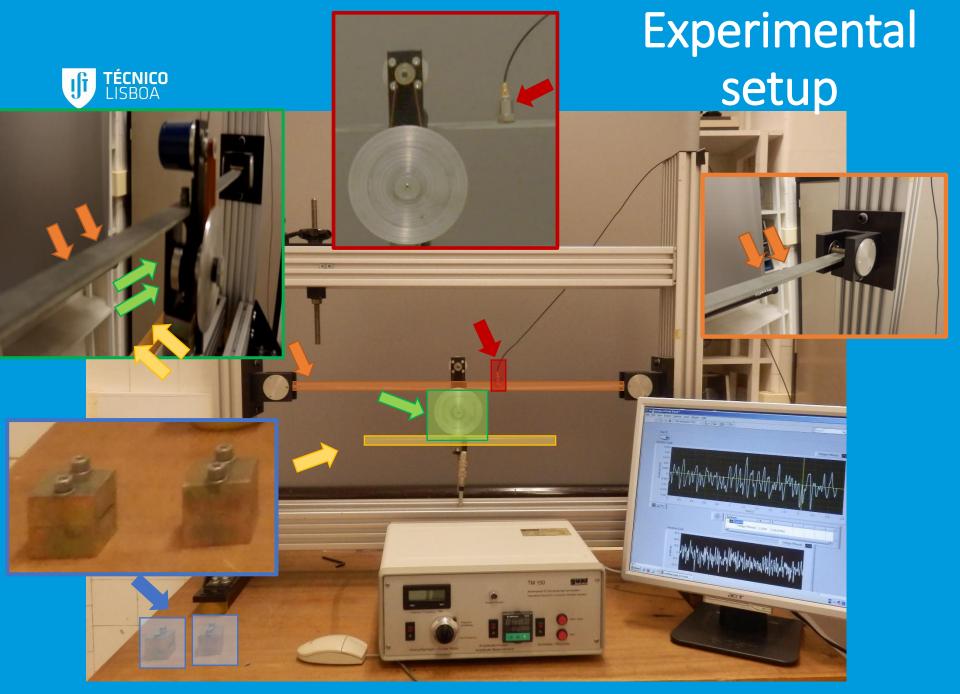
Tuned Mass Dampers (TMDs)

https://www.youtube.com/watch?v=UuTqX5YMGyk

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## Experimental Test Dynamic absorber (Laboratory)



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# Experimental setup



Without absorber	With absorber	With absorber	With absorber
$f_n = 7.34  Hz$	f = 7.34  Hz	$f_1 = 5.83  Hz$	$f_2 = 13.36  Hz$
$Y_i = 3 mV$	$Y_f = 0.15  mV$	$Y_1 = 0.8 mV$	$Y_2 = 6 mV$

0.0

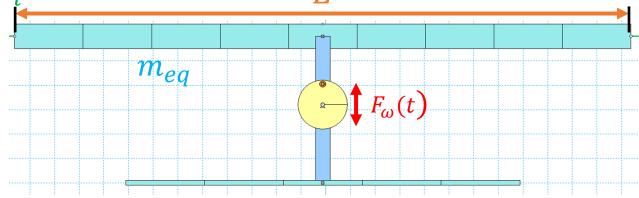


## Experimental Test Dynamic absorber (Simulation)



## Simulation of case I (without absorber)

Objectives: Knowing the parameters: m<sub>eq</sub>, L e f<sub>n</sub> (defined for each group), one obtains the time frequency response of the system under forced vibrations and estimate the values of k<sub>eq</sub>, E and Y<sub>i</sub>.



### **Known parameters**

- $m_{eq}$  Equivalent mass of the main system;
- L Length of the beam;
- $f_n$  1st natural frequency of the system.

### Parameters to be determined

- $k_{eq}$  Equivalent stiffness of the main system;
- E Modulus of elasticity of the main system's beam;
- $Y_i$  Vibration amplitude at the exciting frequency.

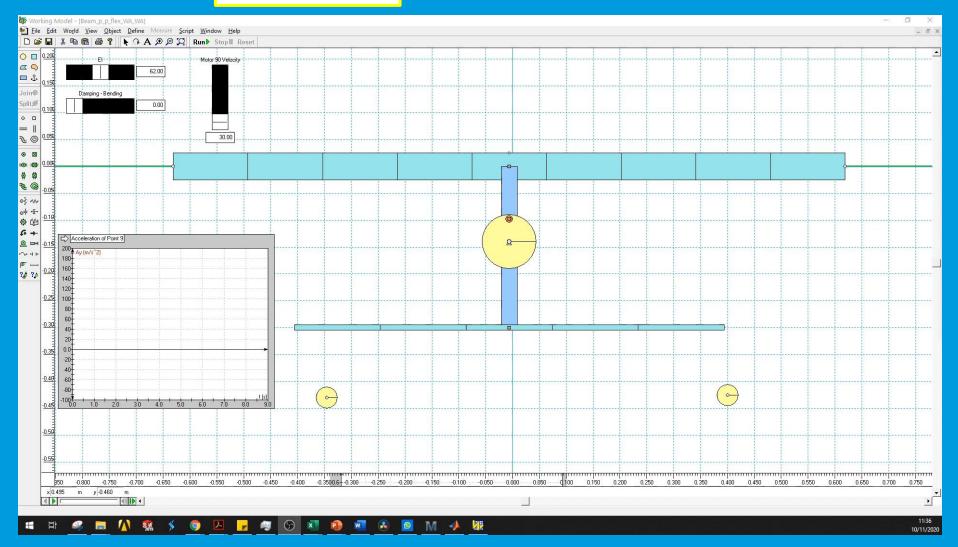


Without absorber

 $f_n = 40 \ rad/s$ 

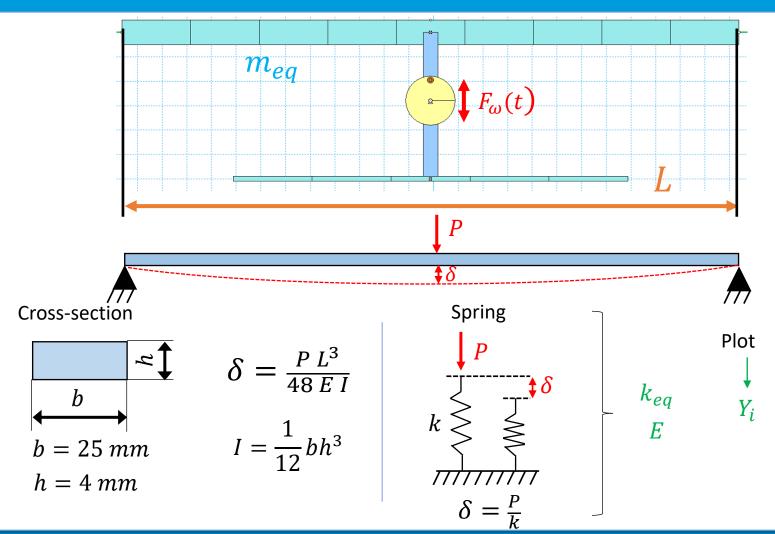
 $Y_i = 180 \text{ m/s}^2$ 

## Simulation of case I (without absorber)



## Simulation of case I (without absorber)

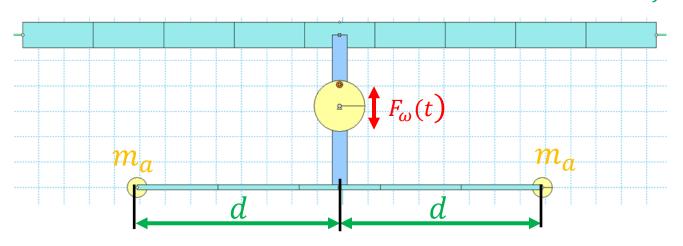






## Simulation of case II (with absorber)

Objectives: Knowing the parameters: m<sub>a</sub> e f<sub>pabs</sub> (defined for each group, one obtains the time frequency response of the system under forced vibrations and estimate the values of d, Y<sub>f</sub> e η<sub>abs</sub>.



### **Known parameters**

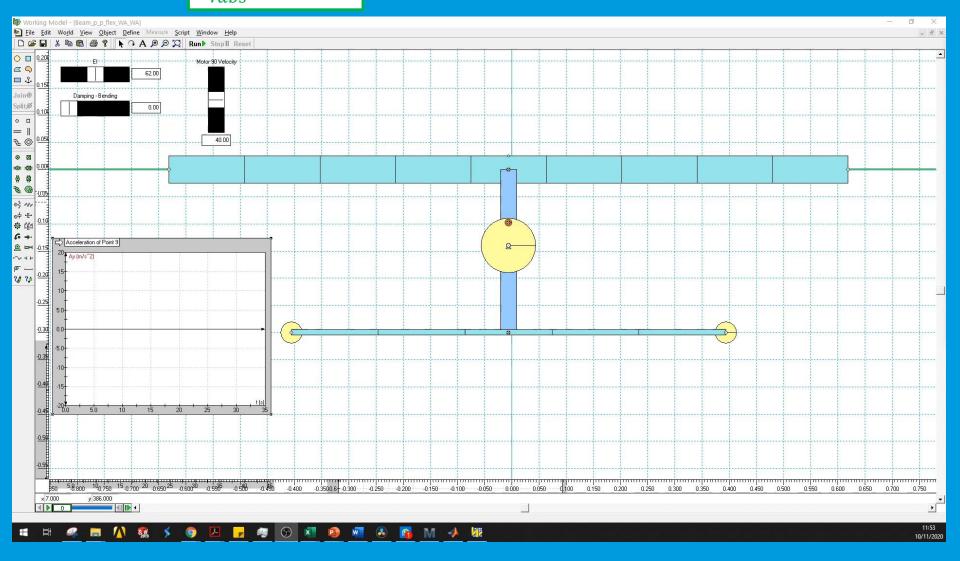
 $m_a$  – Mass of the absorber (each);  $f_{p_{abs}}$ – Excitation frequency at which is intended to suppress vibrations.

### Parameters to be determined

- d Distance of the mass;
- $Y_f$  –Amplitude of vibration at the excitation frequency;
- $\eta_{abs}$  Efficiency of the absorber.

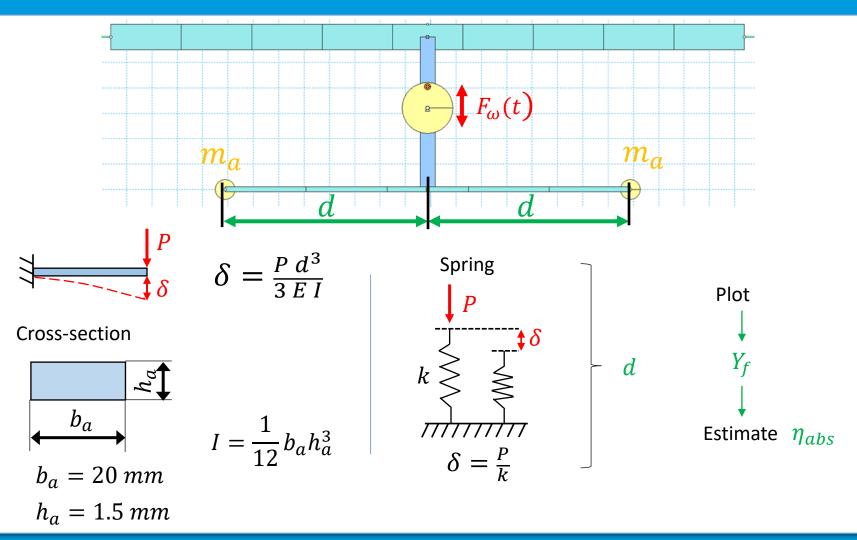


**TÉCNICO**<br/>LISBOAWith absorber<br/> $f_{p_{abs}} = 40 \ rad/s$ <br/> $Y_f = 5 \ m/s^2$ <br/> $\eta_{abs} = 97.2\%$ 



## Simulation of case II (with absorber)







## **Experiment procedures**

- 1) To each group are assigned the following parameters  $m_{eq}$ , L,  $f_n$ ,  $m_a$  and  $f_{p_{abs}}$ ;
- The teacher conducts the simulation of test I and returns to the group the corresponding files that contains the data referring to the time response without the absorber;
- 3) The teacher conducts the simulation of test II and returns to the group the corresponding files that contains the data referring to the time response with the absorber;
- 4) Each group downloads a file where it must be registered the values of all requested parameters (the file can be printed and later scanned in \*pdf format or digitally filled);
- 5) The group must submit the file described in 4).



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